

Multi-Dof systems with damping

$$M \ddot{\vec{x}} + K \vec{x} = B_f f(t) \rightarrow \text{physical coordinates}$$

$$\vec{x} = M^{-1/2} \vec{q} \rightarrow \text{pre multiply by } M^{-1/2}$$

$$M^{-1/2} M M^{-1/2} \ddot{\vec{q}} + M^{-1/2} K M^{-1/2} \vec{q} = M^{-1/2} B_f f(t)$$

= 0 (homogeneous e-values/e-vectors)

$$I \ddot{\vec{q}} + \tilde{K} \vec{q} = 0$$
$$M^{-1} K$$

$M^{-1/2} K M^{-1/2} \rightarrow$ symmetric, positive definite, yields orthogonal eigenvectors

$M^{-1} K \rightarrow$ non-symmetric (symmetric if $m_1 = m_2$)

$$|\tilde{K} - \lambda I| = 0, \lambda_1, \lambda_2$$

e-vectors V_1, V_2 are orthogonal

Create a modal matrix $P = [V_1, V_2]$ Then $P^T P = I$, where V_i 's are orthonormal

$$V_1^T V_1 = 1$$

$$V_1^T V_2 = 0$$

$$M^{-1} K: u_1, u_2$$

-problem: $u_1^T u_2 \neq 0$

spanning the space means $x(t) = \overset{\text{fill in}}{2} u_1 + \overset{\text{fill in}}{0} u_2$

$$\vec{x} = M^{-1/2} \vec{q} \quad \text{Coordinate transformation}$$
$$\vec{q} = M^{1/2} \vec{x}$$

$$\text{Implies: } I \ddot{\vec{q}} + \tilde{K} \vec{q} = M^{-1/2} B_f f(t)$$

$$\text{Another transformation: } \vec{q} = P \vec{r}$$

$$\mathbf{I} \ddot{\mathbf{P}} \tilde{\mathbf{r}} + \tilde{\mathbf{K}} \mathbf{P} \tilde{\mathbf{r}} = \mathbf{M}^{-1/2} \mathbf{B}_f f(t)$$

premultiply by \mathbf{P}^T

$$\mathbf{P}^T \mathbf{I} \ddot{\mathbf{P}} \tilde{\mathbf{r}} + \mathbf{P}^T \tilde{\mathbf{K}} \mathbf{P} \tilde{\mathbf{r}} = \mathbf{P}^T \mathbf{M}^{-1/2} \mathbf{B}_f f(t)$$

$$\mathbf{I} \ddot{\tilde{\mathbf{r}}} + \mathbf{\Delta} \tilde{\mathbf{r}} = \mathbf{P}^T \mathbf{M}^{-1/2} \mathbf{B}_f f(t)$$

$\tilde{\mathbf{r}} =$ modal coordinates

$$\tilde{\mathbf{r}} = \begin{bmatrix} \tilde{r}_1(t) \\ \tilde{r}_2(t) \\ \vdots \end{bmatrix}$$

Solution can be written as a linear combination of mode shapes

Projecting now from physical to modal

Example: $M_1 = 2, M_2 = 1$
 $K_1 = 24, K_2 = 3$ > decouple the system using $\mathbf{M}^{-1} \mathbf{K}$, then symmetric eigenvalue approach
 $\omega_1^2 = 2, \omega_2^2 = 4$

$$\mathbf{I} \ddot{\tilde{\mathbf{r}}} + \mathbf{\Delta} \tilde{\mathbf{r}} = \mathbf{P}_i f(t)$$

$\mathbf{P}_i =$ modal participation factors

$\lambda = 1$  1 mode

$\lambda = 2$ 2 modes, $\lambda = 3$ 3 modes

Damping: in MDoF systems

$$\mathbf{M} \ddot{\mathbf{X}} + \mathbf{D} \dot{\mathbf{X}} + \mathbf{K} \mathbf{X} = \mathbf{B}_f f(t)$$

$$\mathbf{D} = \alpha \mathbf{M} + \beta \mathbf{K}$$

α, β are chosen and allow a form of damping that permits us to decouple the systems

$$\tilde{\mathbf{X}} = \mathbf{M}^{-1/2} \tilde{\mathbf{q}}$$

$$\tilde{\mathbf{q}} = \mathbf{P} \tilde{\mathbf{r}}$$

$$\boxed{\tilde{\mathbf{X}} = \mathbf{M}^{-1/2} \mathbf{P} \tilde{\mathbf{r}}}$$

$$M \ddot{\vec{r}} + D \dot{\vec{r}} + K \vec{r} = B f(t)$$

Assuming we know P
from \tilde{K}

-premultiply by $P^T M^{-1/2}$

$$P^T M^{-1/2} M M^{-1/2} \ddot{\vec{r}} + P^T M^{-1/2} D M^{-1/2} \dot{\vec{r}} + P^T M^{-1/2} K M^{-1/2} \vec{r} = P^T M^{-1/2} B f(t)$$

$$I \ddot{\vec{r}} + \Delta_0 \dot{\vec{r}} + \Delta_K \vec{r} = \vec{F}(t) \quad \vec{r} = \begin{bmatrix} r_1 \\ r_2 \end{bmatrix}$$

$$\begin{aligned} \Delta_0 &= P^T M^{-1/2} D M^{-1/2} P = P^T M^{-1/2} (\alpha M + \beta K) M^{-1/2} P \\ &= \alpha P^T M^{-1/2} M M^{-1/2} P + \beta P^T M^{-1/2} K M^{-1/2} P \end{aligned}$$

$$\Delta_0 = \alpha I + \beta \Delta_K = \text{diag}[2\zeta_i \omega_i]$$

Normal modes: - nodes are fixed
- modes being real numbers not complex

$$\vec{U} = M^{-1/2} \vec{V}_i$$

$$\vec{X} = M^{-1/2} \vec{Q}_i$$

$M^{-1}K$ always gives mode shapes, but they might not
decouple the system